Introductory Motion and Control

Compensator Design for ITAE Optimal Response with Pre-filters

(Reference: Dorf & Bishop, Modern Control Systems, Prentice-Hall, 2001)

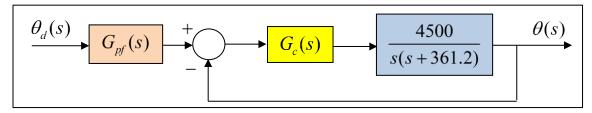
The forms of *ITAE optimal* transfer functions for *step* and *ramp* inputs are given in Tables 5.6 and 5.7 of Dorf & Bishop's Modern Control Systems. To guarantee ITAE optimal behavior, a compensator can be designed to force the closed-loop transfer function to be of *ITAE optimal form*. The use of *pre-filters* may be required to obtain optimal form.

Design Strategy

- 1. *Step Input*: Assume $\zeta = 0.7$; *Ramp Input*: Assume $\zeta = 1.6$
- 2. Then *select* ω_n to satisfy the *settling time* specification.
- 3. Find the compensator parameters to give a *characteristic equation* of the form given in Table 5.6 (step input) or Table 5.7 (ramp input).
- 4. Design a *pre-filter* to "cancel" the unwanted terms in the numerator of the closed-loop transfer function.

Example

Design a PID compensator that will produce *ITAE optimal*, *closed-loop step response* with a settling time $T_s = 0.005$ (sec) for the aircraft attitude control system shown in the block diagram. Use a pre-filter as necessary.



Solution

- 1. Find ω_n by setting $T_s = 4/\zeta \omega_n = 0.005$ (sec). Assuming $\zeta = 0.7$ (for a step input), $\omega_n = 1143$ (rad/sec).
- 2. Assuming a PID controller of the form $G_c(s) = \frac{K(s^2 + as + b)}{s}$, the closed loop transfer function (without the pre-filter) is found to be

$$T(s) = \frac{4500K(s^2 + as + b)}{s^2(s + 361.2) + 4500K(s^2 + as + b)}$$

With *PID* control and *no pre-filter*, this system is *type-2*, so it can be *optimized* for *ramp input*. However, in this example, the *optimization* is for *step response*.

3. Equate the *characteristic equation* with that provided by Table 5.6 with $\omega_n = 1143$.

$$s^{3} + (361.2 + 4500K)s^{2} + (4500aK)s + (4500bK)$$
$$= s^{3} + 1.75\omega_{n}s^{2} + 2.15\omega_{n}^{2}s + \omega_{n}^{3}$$

Equating the polynomial coefficients and solving for K, a, and b gives

or
$$K = 0.36423$$
 $a = 1713.73$ $b = 9.110673 \times 10^5$ $K_p = 624.2$ $K_I = 331,838$ $K_D = 0.36423$

4. *Pre-filter*: To provide ITAE optimal response, choose the pre-filter

$$G_{pf}(s) = \frac{(1143)^3}{1639(s^2 + as + b)}$$

So, the final transfer function has ITAE optimal form

$$\frac{\theta_d(s)}{\theta(s)} = \left(\frac{(1143)^3}{1639(s^2 + as + b)}\right) \left(\frac{1639(s^2 + as + b)}{s^3 + 2000s^2 + (2.8089 \times 10^6)s + (1143)^3}\right)$$

$$\Rightarrow \left[\frac{\theta_d(s)}{\theta(s)}\right] = \frac{(1143)^3}{s^3 + 2000s^2 + (2.8089 \times 10^6)s + (1143)^3}$$

- 5. Closed-loop step response *with* and *without* pre-filter: From the plots below, it is clear the closed loop system *without a pre-filter* has *large overshoot* due to the zeros in the PID compensator. This problem is removed by including a pre-filter. Both systems have settling times slightly larger than 0.005 (sec). Further increases on the choice of ω_n will resolve this issue as well.
- 6. Error analysis with a pre-filter: The system error can be calculated as follows

$$E(s) = \theta_d(s) - \theta(s) = \theta_d(s) - \left(G_{pf}(s)T(s)\right)\theta_d(s) = \left(1 - G_{pf}(s)T(s)\right)\theta_d(s)$$

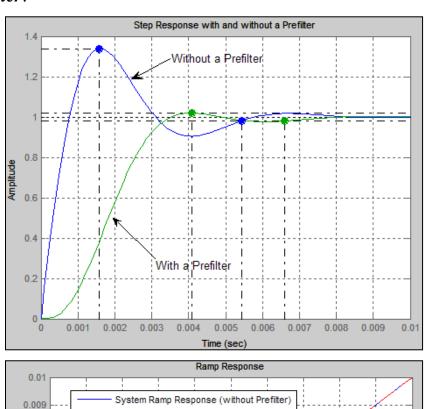
$$= \left(1 - \frac{(1143)^3}{s^3 + 2000s^2 + (2.8089E6)s + (1143)^3}\right)\theta_d(s)$$

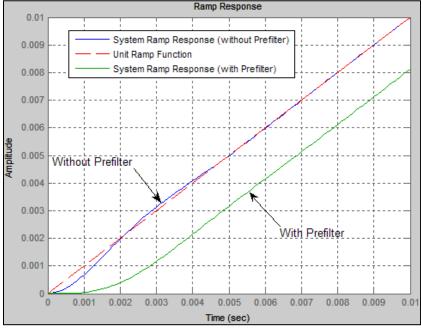
So, the error transfer function is

$$\frac{E(s)}{\theta_d(s)} = \frac{s(s^2 + 2000s + 2.8089E6)}{s^3 + 2000s^2 + (2.8089E6)s + (1143)^3}$$

Note the system *with the pre-filter* has a *zero* steady-state error to a *step input* and a *finite* steady-state error to a *ramp input* (typical of a *type-1* system).

7. Error analysis with no pre-filter: The system with no pre-filter is a type-2 system, so it has a zero steady-state error to both step and ramp inputs. Clearly, some error control is lost when using the pre-filter.





Kamman – Introductory Motion and Control – Compensator Design for ITAE Optimal Response with Pre-filters – page: 3/3